

## Unit 5 - State variable Analysis.

### Introduction

- Analysis & design of linear/nonlinear, Time variant / Invariant, MIMO system was carried using state space method.

→ State space analysis is a modern approach & easier for analysis using digital computers.

- Conventional mtd of analysis is to find of system. Drawback of to find

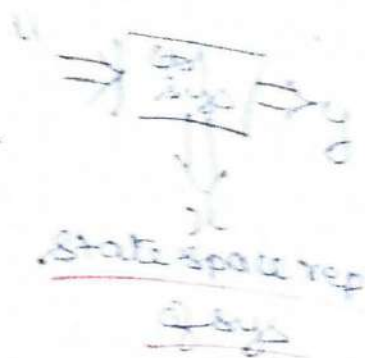
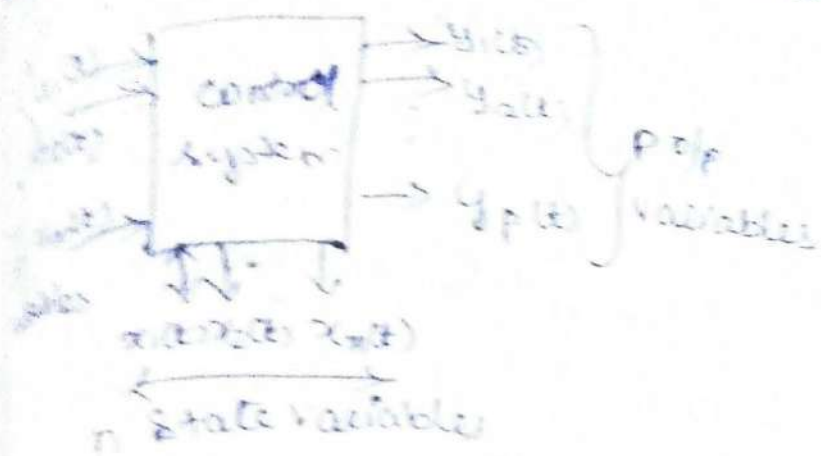
- 1) It is defined under initial condns
- 2) It is applicable to linear time invariant sys
- 3) It is restricted to SISO systems
- 4) Doesn't provide info reg Internal state of state

### Adv of State variable analysis

- 1) Applied for any type of sys.
- 2) Analysis can be carried with initial condns & can be carried on MIMO sys.

### State variable

- A set of variables which describe the sys at any time instant.



input vector

$$u(t) = \begin{bmatrix} u_1(t) \\ u_2(t) \\ \vdots \\ u_m(t) \end{bmatrix}$$

output vector

$$y(t) = \begin{bmatrix} y_1(t) \\ y_2(t) \\ \vdots \\ y_p(t) \end{bmatrix}$$

State variable vector

$$x(t) = \begin{bmatrix} x_1(t) \\ x_2(t) \\ \vdots \\ x_n(t) \end{bmatrix}$$

State eqn rep

$$\dot{x}(t) = \frac{dx}{dt} = f[x(t), u(t)]$$

State model of Linear system

- consists of state eqn & output eqn

- State eqn of a sys is a set of state variables & i/p

$$\dot{x}(t) = f(x(t), u(t))$$

for LTI system 1st derivative of state variables can be expressed as linear combination of state variables & i/p



$$\dot{x}_1 = a_{11}x_1 + a_{12}x_2 + \dots + a_{1n}x_n + b_{11}u_1 + b_{12}u_2 + \dots + b_{1m}u_m$$

$$\dot{x}_2 = a_{21}x_1 + a_{22}x_2 + \dots + a_{2n}x_n + b_{21}u_1 + b_{22}u_2 + \dots + b_{2m}u_m$$

$$\vdots$$

$$\dot{x}_n = a_{n1}x_1 + a_{n2}x_2 + \dots + a_{nn}x_n + b_{n1}u_1 + b_{n2}u_2 + \dots + b_{nm}u_m$$

$a, b$  const.

matrix rep

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \vdots \\ \dot{x}_n \end{bmatrix} = \begin{bmatrix} a_{11} & a_{12} & \dots & a_{1n} \\ a_{21} & a_{22} & \dots & a_{2n} \\ a_{31} & a_{32} & \dots & a_{3n} \\ \vdots & \vdots & \ddots & \vdots \\ a_{n1} & a_{n2} & \dots & a_{nn} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ \vdots \\ x_n \end{bmatrix} + \begin{bmatrix} b_{11} & b_{12} & \dots & b_{1m} \\ b_{21} & b_{22} & \dots & b_{2m} \\ b_{31} & b_{32} & \dots & b_{3m} \\ \vdots & \vdots & \ddots & \vdots \\ b_{n1} & b_{n2} & \dots & b_{nm} \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ \vdots \\ u_m \end{bmatrix}$$

order (n x n)      n x 1      n x m      m x 1

matrix eqn

$$\dot{x}(t) = A x(t) + B u(t) \rightarrow \text{state eqn}$$

of LTI system

o/p at any time is fn of state variables & i/p

$$\text{o/p vector } y(t) = f(x(t), u(t))$$

$$y_1 = c_{11}x_1 + c_{12}x_2 + \dots + c_{1n}x_n + d_{11}u_1 + d_{12}u_2 + \dots + d_{1m}u_m$$

$$y_2 = c_{21}x_1 + c_{22}x_2 + \dots + c_{2n}x_n + d_{21}u_1 + d_{22}u_2 + \dots + d_{2m}u_m$$

$$\vdots$$

$$y_p = c_{p1}x_1 + c_{p2}x_2 + \dots + c_{pn}x_n + d_{p1}u_1 + d_{p2}u_2 + \dots + d_{pm}u_m$$

$c, d \rightarrow \text{const.}$

$$y(t) = C x(t) + D u(t) \rightarrow \text{o/p eqn of LTI sys}$$

order  $\Rightarrow$       p x 1      p x n      n x 1      p x m      m x 1

$$\begin{bmatrix} c_{11} & c_{12} & \dots & c_{1n} \\ c_{21} & c_{22} & \dots & c_{2n} \\ \vdots & \vdots & \ddots & \vdots \\ c_{p1} & c_{p2} & \dots & c_{pn} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ \vdots \\ x_n \end{bmatrix} + \begin{bmatrix} d_{11} & d_{12} & \dots & d_{1m} \\ d_{21} & d_{22} & \dots & d_{2m} \\ \vdots & \vdots & \ddots & \vdots \\ d_{p1} & d_{p2} & \dots & d_{pm} \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ u_3 \\ \vdots \\ u_m \end{bmatrix}$$

state diag

- pictorial rep of state model of sys is called state diagram.

- state diag of sys can be either block diag or in signal flow graph form.

- state diag describes the relationship b/w among state variables & provides physical interpretation of state variables.

- Time domain state diag is obtained from diff eqn governing system & this diag can be used for simulation of sys in analog comp.

- s domain state diag can be obtained from tr. fn of system.

- state diag provides direct relationship b/w time & s-domain.

- State diagram of a state model is constructed using 3 basic elements  
Scalar, adder & Integrator

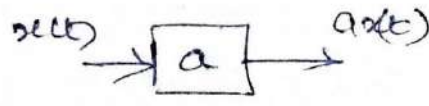
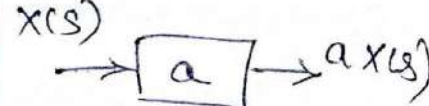
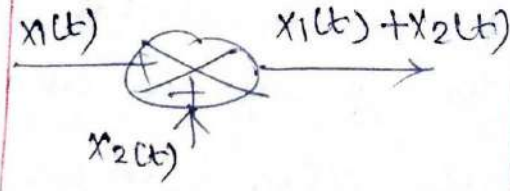
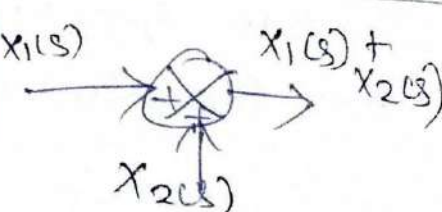
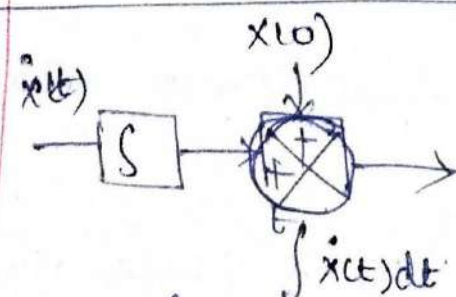
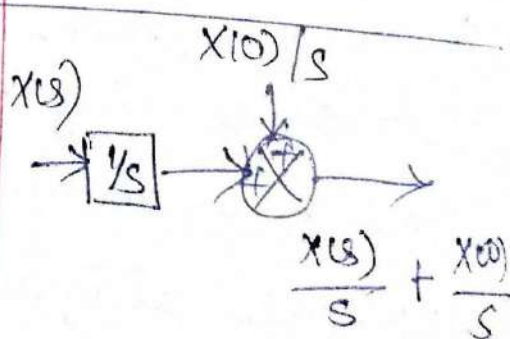


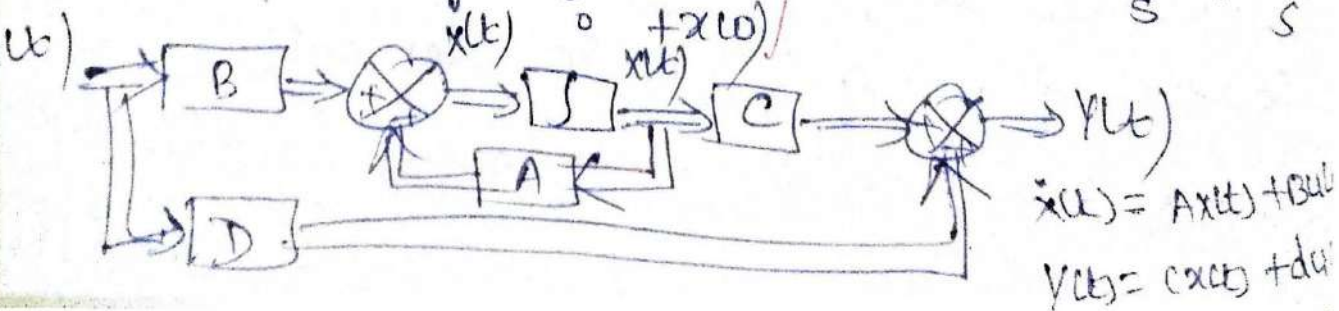
**Scalar** - used to multiply a signal by a const. If  $x(t)$  is multiplied by a scalar 'a' to give  $o/p(x(t) \cdot a)$

**Adder** : Adds 2 or more signals. o/p of adder is sum of incoming signals

**Integrator** : Integrates the signal, used to integrate the derivatives of state variables to get state variables. Initial condns of state variables can be added by using an adder after integrator.

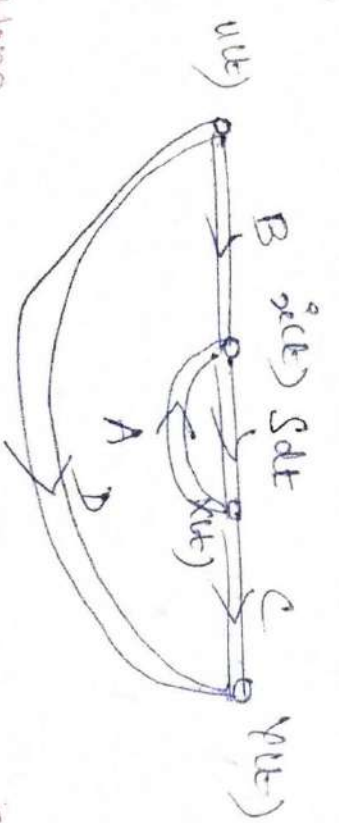
### Elements of block diagram

Element	time domain	s-domain
Scalar		
Adder		
Integrator		



# elements of signal-flow graph

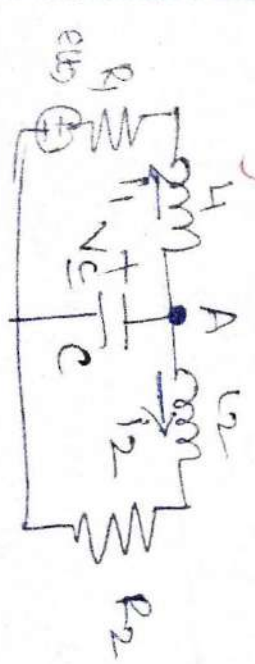
element	time domain	s-domain
Summation	$x_1(t) \rightarrow \oplus \rightarrow x_2(t)$	$x_1(s) \rightarrow \oplus \rightarrow x_2(s)$
Gain	$x_1(t) \xrightarrow{a} x_2(t)$	$x_1(s) \xrightarrow{a} x_2(s)$
Integrator	$x_1(t) \xrightarrow{\int} x_2(t)$	$x_1(s) \xrightarrow{1/s} x_2(s)$



problem

1) Obtain state model of

choosing min no. of state variables



$x_1 = i_1 = \int u(t) dt$   
 $x_2 = i_2 = \int u(t) dt$   
 $x_3 = v_C = \text{voltage across } C$

At A,  $KCL, i_1 + i_2 + C \frac{dv_C}{dt} = 0$

Sub state variables

$$x_1 + x_2 + C \dot{x}_3 = 0$$

$$C \dot{x}_3 = -x_1 - x_2$$

$$\dot{x}_3 = -\frac{1}{C} x_1 - \frac{1}{C} x_2$$

KVL at loop 1

$$e(t) + i_1 R_1 + L_1 \frac{di_1}{dt} = V_C$$

Sub state variables

$$e(t) + x_1 R_1 + L_1 \dot{x}_1 = x_3$$

use = current 'ip' to eqn

$$u + x_1 R_1 + L_1 \dot{x}_1 = x_3$$

$$L_1 \dot{x}_1 = x_3 - x_1 R_1 - u$$

$$\dot{x}_1 = -\frac{R_1}{L_1} x_1 + \frac{1}{L_1} x_3 - \frac{1}{L_1} u$$

eg KVL at loop 2,

$$V_C = L_2 \frac{di_2}{dt} + i_2 R_2,$$

$$\dot{x}_3 = L_2 \dot{x}_2 + x_2 R_2$$







## Homogeneous eqn

If  $A$  is a const matrix & i/p ctrl forces are zero then eqn takes form  $\dot{X}(t) = AX(t)$

This is homogeneous eqn.

- If i/p is zero, in such sys, driving forces is provided by initial condns of sys to produce o/p.

Eg. consider a RC ckt in which capacitor is initially charged to  $V$  volts. I is o/p. Now there is no i/p ctrl force ie ext voltage applied to sys. But initial voltage on capacitor drives the current thro' sys & capacitor starts discharging thro'  $R$ . Such a sys which works on initial condns w/o any i/p applied to it is called homogeneous sys.

## Non homogeneous eqn

If  $A$  is a const matrix & matrix  $U(t)$  is non zero vector ie i/p ctrl forces are applied to sys then eqn takes normal form as,

$$\dot{X}(t) = AX(t) + BU(t) \rightarrow \text{Non homogeneous eqn}$$

- most practical sys req i/p to drive them.

Such sys are non homogeneous linear sys. The solution of state eqn is obtained by considering basic mtd of finding soln of homogeneous eqn

diagonalizing transformation

$$\dot{x} = Ax + Bu, \quad y = Cx + Du, \quad x(0) = x_0$$

$$\hat{x} = Qx \quad (Q = \text{transformation matrix})$$

$$x = Q^{-1}\hat{x}$$

$$\dot{\hat{x}} = \Lambda \hat{x} + B'u, \quad y = C'\hat{x} + Du$$

$$\Lambda = \begin{bmatrix} \lambda_1 & 0 & 0 \\ 0 & \lambda_2 & 0 \\ 0 & 0 & \lambda_n \end{bmatrix}$$

• by doing the diagonalizing transformation, nothing is in the  $u(s)$  &  $y(s)$  will not be altered

• If  $\hat{x}_n = 0$ , then  $x_n(t)$  is uncontrollable by the  $u(t)$ , since  $x_n(t)$  is characterized by mode  $e^{\lambda_n t}$ . By the  $n$ -th,  $x_n(t) = e^{\lambda_n t} x_n(0)$

• The lack of controllability of the state  $x_n(t)$  is due to a zero  $n$ -th row of  $B$  is  $b_n = 0$ , which would cause a complete zero row in the controllability matrix:

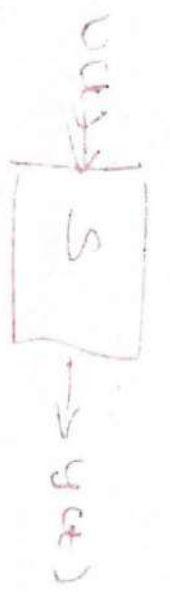
$$[A, B] = \begin{bmatrix} A & B \\ A^2 & AB \\ A^3 & A^2B \\ \vdots & \vdots \end{bmatrix}$$

$$= \begin{bmatrix} b_1' & \lambda_1 b_1' & \lambda_1^2 b_1' & \dots & \lambda_1^{n-1} b_1' \\ b_2' & \lambda_2 b_2' & \lambda_2^2 b_2' & \dots & \lambda_2^{n-1} b_2' \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ b_n' & \lambda_n b_n' & \lambda_n^2 b_n' & \dots & \lambda_n^{n-1} b_n' \end{bmatrix}$$

—  $C(A, b)$  matrix with all non zero row has rank of  $N$ .

— In fact  $B = \mathcal{O}^T B$  or  $B = \mathcal{O} B$ , Thus a non singular  $C(A, b)$  matrix implies a non-singular matrix  $C(A, b) = \begin{bmatrix} B & AB & A^2B & \dots & A^{n-1}B \end{bmatrix}$

Tr-Fn from state variable rep



General form of LTI system

$$\frac{d^n y}{dt^n} + a_{n-1} \frac{d^{n-1} y}{dt^{n-1}} + \dots + a_0 y(t) = b_{n-1} \frac{d^{n-1} u}{dt^{n-1}} + \dots + b_0 u(t)$$

This model has the property

$u(t) = a_1 u(t) + a_2 u(t) \Rightarrow y(t) = a_1 y(t) + a_2 y(t)$  satisfies general form.

— Assume sys is at rest prior to time  $t_0 = 0$ , & if  $u(t)$  (cost  $< \infty$ ) produces the o/p  $y(t)$  (cost can be rep by a trfn in term of LT variables)

$$y(s) = \frac{b_n s^n + b_{n-1} s^{n-1} + \dots + b_0}{a_n s^n + a_{n-1} s^{n-1} + \dots + a_0} u(s)$$



Then applying the same i/p shifted by any  
 int'l of time produces same o/p shifted by  
 same amt of time.

$$y(s) = \left( \frac{b_m s^m + b_{m-1} s^{m-1} + \dots + b_0}{a_n s^n + a_{n-1} s^{n-1} + \dots + a_0} \right) e^{-as} u(s)$$

having all  $b_i = 0$  ( $i > 0$ ), a state space rep  
 arose out of a reduction to a sys of 1<sup>st</sup> order  
 diff eqns.

$$y^{(n)} = f(t, u(t), y, \dot{y}, \ddot{y}, \dots, y^{(n-1)})$$

with initial condns  $y(0) = y_0, \dot{y}(0), \dots, y^{(n-1)}(0) = y_{n-1}^{(0)}$

consider a vector  $x \in \mathbb{R}^n$  with  $x_1 = y, x_2 = \dot{y}, x_3 = \ddot{y}, \dots$   
 $x_n = y^{(n-1)}$

$$\frac{d}{dt} X = \begin{pmatrix} x_1 \\ x_2 \\ \vdots \\ x_n \\ f(t, u(t), y, \dot{y}, \ddot{y}, \dots, y^{(n-1)}) \end{pmatrix}$$

n case of linear sys

$$\frac{d}{dt} X = \begin{pmatrix} 0 & 1 & 0 & \dots & 0 \\ 0 & 0 & 1 & 0 & \dots & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots & \vdots \\ 0 & 0 & \dots & \dots & 1 \\ -a_0 & -a_1 & \dots & \dots & -a_{n-1} \end{pmatrix} X + \begin{pmatrix} 0 \\ 0 \\ 0 \\ \vdots \\ 1 \end{pmatrix} u(t)$$

$$y(t) = \begin{pmatrix} 1 & 0 & 0 & \dots & 0 \end{pmatrix} X$$

General form

$$\frac{d}{dt} X = \begin{bmatrix} 0 & 1 & 0 & \dots & 0 \\ 0 & 0 & 1 & 0 & \dots & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots & \vdots \\ 0 & 0 & \dots & \dots & 1 \\ -a_0 & -a_1 & \dots & \dots & -a_m \end{bmatrix} X + \begin{bmatrix} 0 \\ 0 \\ \vdots \\ 0 \\ 1 \end{bmatrix} u(t)$$

$$y(t) = [b_0 \ b_1 \ \dots \ b_m \ 0 \ \dots \ 0] X$$

Abbreviation

$$\dot{X} = AX + BU, \quad y = CX + DU \quad ; D=0.$$

Controller canonical form of sys

State space rep for discrete timesys

The dynamics of a linear time shift invariant DT sys may be expressed in terms of state (plant) eqn & o/p (observation or measurement) eqn as follows.

$$x(k+1) = Ax(k) + Bu(k),$$

$$y(k) = Cx(k) + Du(k)$$

where  $x(k)$  is an  $n$  dimensional state vector at time  $= kT$ .

$u(k) \rightarrow$   $i$  dimensional ctrl i/p vector.

$y(k) =$   $m$  dimensional o/p vector.

$$x(k) = [x_1(k), x_2(k), \dots, x_n(k)]^T$$

$$u(k) = [u_1(k), u_2(k), \dots, u_n(k)]^T$$

$$y(k) = [y_1(k), y_2(k), \dots, y_m(k)]^T$$

The parameters of  $A$ , an  $n \times n$  (plant parameter) matrix,  $B$  an  $n \times r$  ctrl i/p matrix &  $C$  an  $m \times r$  o/p parameter matrix,  $D$  an  $m \times r$  parameter matrix. All are const for an LTI sys.

State variable rep of SISO DT sys

$$x(k+1) = Ax(k) + bu(k)$$

$$y(k) = e^T x(k) + du(k)$$

- where i/p  $u$ , o/p  $y$  &  $d$  are scalars,  $b$  &  $c$  are  $n$  dimensional vectors.

- The concepts of controllability & observability for DT sys are similar to continuous timesys

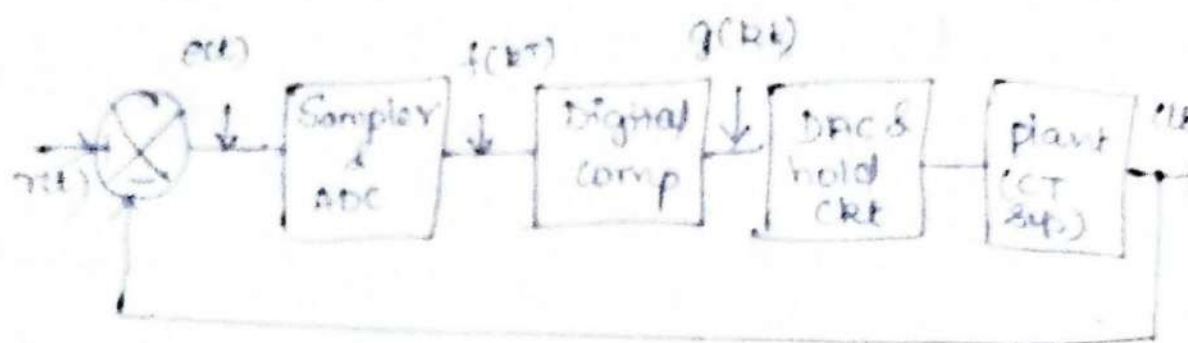
- A DT sys is said to be controllable if there exists a finite integer  $n$  & i/p  $u(k)$ ;

$k \in [0, n-1]$  that will transfer any state  $x^0$  to state  $x^n$  at  $k=n$ .



## Sampled Data sys

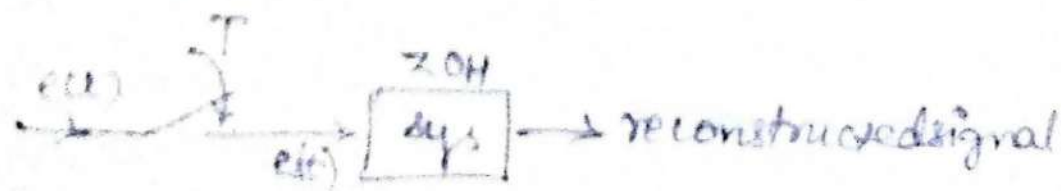
- when the signal is inf at any or discrete pts in a sys is in form of discrete pulses. Then sys is called discrete data sys. In cbl sys engg the discrete data sys is popularly known as sampled data sys



## Sampling theorem

A Band limited CT signal with highest freq  $(f_m)$  Hz can be uniquely recovered from its samples provided that the sampling rate  $F_s$  is greater than or equal to  $2f_m$  samples per sec.  $f_s \geq 2f_m$

## Sample & hold

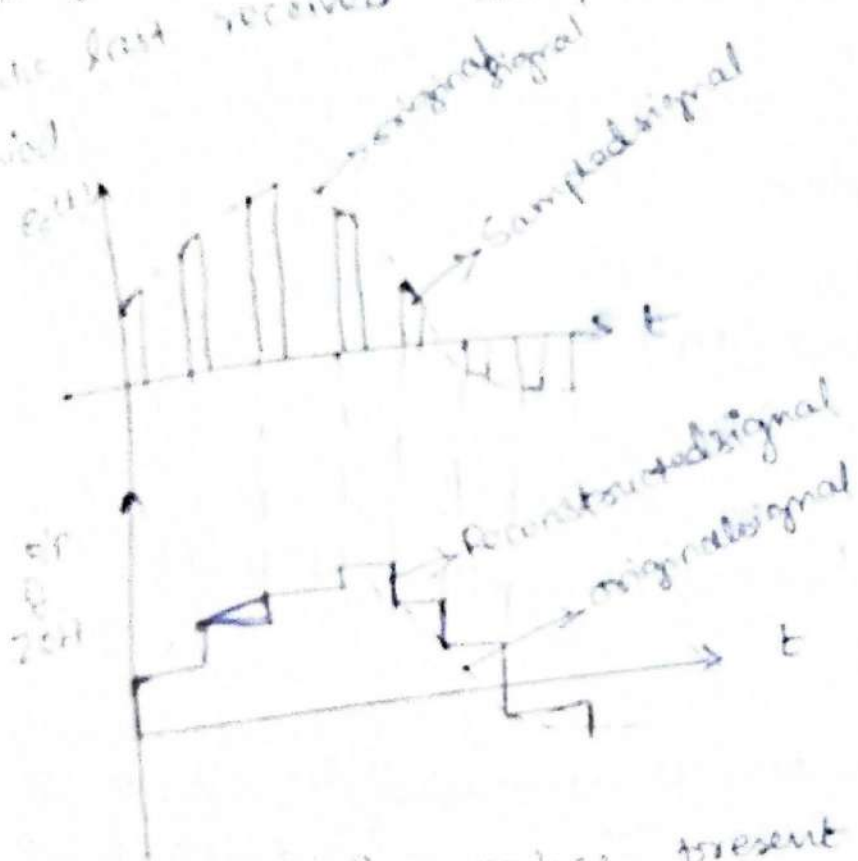


- signal go to digital controller is a

sampled data signal & to run the controller gives the controller output in dig form. But the sys. to be controlled needs an analog cbl.

as  $\frac{1}{p}$ .  $\therefore$  the dig o/p of controllers must  
converters into analog form.

This can be achieved by means of various  
types of hold ckt. The simplest hold ckt are  
zero order hold (ZOH). In ZOH, the reconstructed  
analog signal acquires the same values as  
the last received sample for entire sampling  
period.



The high freq. noises present in reconstructed  
signal are automatically filtered out by its delay  
component which behaves like low pass filters.  
In  $1^{st}$  order hold, last 2 signals for current  
sampling period. It is higher order hold ckt  
has no particular adv over ZOH.

## Unit 2 matlab pgm

1) write matlab pgm to find unit step response of fol. sys  $M(s) = 4 / (s^2 + 5s + 4)$

clc

syms s complex;

$$R = 1/s;$$

$$m = 4 / (s^2 + 5*s + 4);$$

$$S1 = R * M$$

dist ('unit step response of system is');

$$s2 = \text{ilaplace}(S1)$$

$$t = 0 : 0.005 : 10;$$

~~step(m)~~

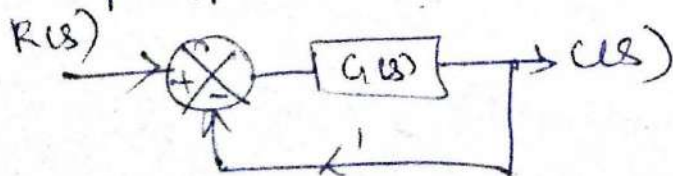
plot(t, s2);

xlabel('time, t in sec');

ylabel('unit step response s2(t)');

## problem

1) unity FB system is characterised by an open loop trfn  $G(s) = \frac{K}{s(s+1)}$ . Determine gain  $K$ , so that sys will have damping ratio of 0.5 for this value of  $K$ . Determine peak overshoot & time at peak overshoot for a unit step i/p.



$$\frac{C(s)}{R(s)} = \frac{G(s)}{1 + G(s)}$$



$$\frac{C(s)}{R(s)} = \frac{\frac{k}{s(s+10)}}{1 + \frac{k}{s(s+10)}} = \frac{k}{s^2 + 10s + k}$$

compare to fn with std form of 2<sup>nd</sup> order to fn

$$\frac{C(s)}{R(s)} = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} = \frac{k}{s^2 + 10s + k}$$

$$\omega_n^2 = k$$

$$2\zeta\omega_n = 10$$

$$k = 100$$

$$\omega_n = 10 \text{ rad/sec}$$

$$\omega_n = \sqrt{k}$$

$$\zeta = 0.5$$

$$\omega_n = \sqrt{k}$$

$$\sqrt{k} = 10$$

$$\therefore k = 100$$

$$\%M_p = e^{-\frac{\zeta\pi\sqrt{1-\zeta^2}}{\omega_n}} \times 100 = \underline{16.3\%}$$

$$\text{peak time } t_p = \frac{\pi}{\omega_d} = \frac{\pi}{\omega_n \sqrt{1-\zeta^2}} = \underline{0.363 \text{ sec}}$$

open loop to fn of unity FB system is

$$G(s) = \frac{k}{s(sT+1)}, \text{ where } k \& T \text{ are constant}$$

By what factors should the amp gain  $k$  be reduced, so that peak overshoot of unit step response of sys is reduced from 75% to 25%.

$$\frac{C(s)}{R(s)} = \frac{G(s)}{1+G(s)} = \frac{\frac{k/T}{s^2 + \frac{1}{T}s + \frac{k}{T}}}{\frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}}$$

$$\omega_n^2 = k/T$$

$$2\zeta\omega_n = \frac{1}{T}$$

$$\omega_n = \sqrt{k/T}$$

$$\zeta = \frac{1}{2\omega_n T} = \frac{1}{2\sqrt{\frac{k}{T}} \cdot T} = \frac{1}{2\sqrt{kT}}$$

When  $M_p = 0.75$  let  $\xi = \xi_1$  &  $k = k_1$

$m_p = 0.25$  let  $\xi = \xi_2$  &  $k = k_2$

$$M_p = e^{\frac{-\xi\pi}{\sqrt{1-\xi^2}}}$$

take  $\ln$ ,

$$\ln M_p = \frac{-\xi\pi}{\sqrt{1-\xi^2}}$$

Squaring

$$(\ln M_p)^2 = \frac{\xi^2 \pi^2}{1-\xi^2}$$

$$(1-\xi^2)(\ln M_p)^2 = \xi^2 \pi^2$$

$$(\ln M_p)^2 - \xi^2 (\ln M_p)^2 = \xi^2 \pi^2$$

$$(\ln M_p)^2 = \xi^2 \left[ \pi^2 + (\ln M_p)^2 \right]$$

$$\xi^2 = \frac{(\ln M_p)^2}{\pi^2 + (\ln M_p)^2} \rightarrow (1)$$

$$\xi = \frac{1}{\sqrt{4KT}}, \quad \xi^2 = \frac{1}{4KT} \rightarrow (2)$$

$$(1) = (2)$$

$$\frac{1}{4KT} = \frac{(\ln m_p)^2}{\pi^2 + (\ln m_p)^2}$$

$$\frac{1}{K} = \frac{4T (\ln m_p)^2}{\pi^2 + (\ln m_p)^2}$$

$$K = \frac{\pi^2 + (\ln m_p)^2}{4T (\ln m_p)^2}$$

$$K = K_1, m_p = 0.75, \quad K_1 = \frac{\pi^2 + (\ln 0.75)^2}{4T (\ln 0.75)^2} = \frac{30.06}{T}$$



$$T_d = \frac{1}{\omega_p} = 0.25$$

$$\eta = \frac{1}{\omega_p} = \frac{1}{0.25} = 4$$

$$\frac{1}{4T} = \frac{1}{4 \times 0.25} = 1$$

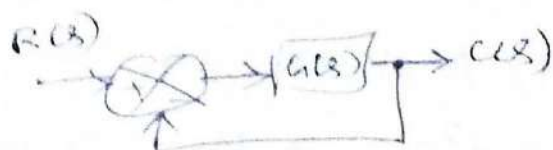
$$T_d = 19.6 \times 20$$

$\therefore K$  should be reduced by 19.6 times

A unity FB ctrl sys has an open loop TF  
 $G(s) = \frac{10}{s(s+2)}$ . Find  $\zeta$ , %MP,  $t_p$  &  $t_s$  for

step IP of 12 units

$$H(s) = 1$$



$$\frac{C(s)}{R(s)} = \frac{G(s)}{1 + G(s)H(s)}$$

$$= \frac{10}{s^2 + 2s + 10} = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

Comparing

$$\omega_n^2 = 10$$

$$\omega_n = \sqrt{10}$$

$$= 3.162 \text{ rad/sec}$$

$$2\zeta\omega_n = 2$$

$$\zeta = \frac{2}{2\omega_n} = 0.316$$

$$\theta = \tan^{-1}\left(\frac{\sqrt{1-\zeta^2}}{\zeta}\right) = 71.5^\circ \times \frac{\pi}{180} = 1.249 \text{ rad}$$

$$\omega_d = \omega_n \sqrt{1-\zeta^2} = 3 \text{ rad/sec}$$

$$t_d = \frac{\pi - \theta}{\omega_d} = 0.63 \text{ sec}$$

$$\%MP = e^{-\frac{\zeta\pi}{\sqrt{1-\zeta^2}}}$$

$$= 35.12\%$$



$$\begin{aligned} \text{Peak overshoot for step i/p of 12 units} &= \frac{\% M_p}{100} \times 12 \text{ units} \\ &= \underline{4.2144 \text{ units}} \end{aligned}$$

$$\text{For } 5\% \text{ error } T_s = 3T = \underline{3 \text{ sec}}$$

$$2\% \text{ error } T_s = 4T = \underline{4 \text{ sec}}$$

$$T = \frac{1}{\xi \omega_n} = \underline{1 \text{ sec}}$$

$$t_p = \frac{\pi}{\omega_d} = \frac{\pi}{3} = \underline{1.047 \text{ sec}}$$

4) A closed loop servo is gov by diff eqn

$\frac{d^2 c}{dt^2} + 8 \frac{dc}{dt} = 64 e$ , where  $c$  is the displacement of o/p shaft,  $e$  is displacement of i/p shaft &  $e = r - c$ . Determine undamped natural freq, damping ratio &  $\% M_p$  for unit step i/p.

$$\text{put } e = r - c$$

$$\frac{d^2 c}{dt^2} + 8 \frac{dc}{dt} = 64 (r - c)$$

Take LT

$$s^2 c(s) + 8s c(s) = 64 [R(s) - c(s)]$$

$$c(s) [s^2 + 8s + 64] = 64 R(s)$$

$$\frac{c(s)}{R(s)} = \frac{64}{s^2 + 8s + 64} = \frac{\omega_n^2}{s^2 + 2\xi\omega_n s + \omega_n^2}$$

$$\omega_n^2 = 64$$

$$\boxed{\omega_n = 8 \text{ rad/sec}}$$

$$2\xi\omega_n = 8$$

$$\boxed{\xi = 0.5}$$

$$\% M_p = e$$

$$\boxed{\% M_p = 16.3\%}$$

Consider a unity feedback system with a closed loop transfer function  $T(s)$ .  
 The error signal  $E(s)$  is given by  $E(s) = \frac{1}{1 + T(s)}$ .  
 Show that steady state error with unit ramp  $\frac{1}{s}$  is given by  $\frac{1}{K_v}$ .

Proof:

$$\frac{E(s)}{K_v} = \frac{E(s)}{s \cdot 1} = \lim_{s \rightarrow 0} s E(s)$$

$$E(s) = \frac{1}{1 + T(s)}$$

$$\lim_{s \rightarrow 0} s E(s) = \lim_{s \rightarrow 0} \frac{s}{1 + T(s)}$$

$$= \frac{\lim_{s \rightarrow 0} s}{\lim_{s \rightarrow 0} (1 + T(s))} = \frac{0}{1 + K_v} = 0$$

$$\boxed{E(s) = \frac{K_v}{s(s + K_v)}}$$

unit ramp  $\frac{1}{s}$ ,  $K_v = \lim_{s \rightarrow 0} s G(s)H(s)$

$$= \lim_{s \rightarrow 0} \frac{K_v}{s(s + K_v)}$$

$$K_v = \frac{1}{\alpha - \beta}$$

$$\boxed{E_{ss} = \frac{1}{K_v} = \frac{\alpha - \beta}{1}} \text{ Hence proved}$$

b) A unity FB system has fwd to FGCS =  $K_1(2s+1)$   
 i/p  $x(t) = 1+6t$ , determine  $K_1$  so that  $\text{SSE} < 0.1$

$$R(s) = \mathcal{L}\{x(t)\} = \frac{1}{s} + \frac{6}{s^2}$$

$$E(s) = \frac{R(s)}{1+G(s)H(s)} = \frac{\frac{1}{s} + \frac{6}{s^2}}{1 + \frac{K_1(2s+1)}{s(5s+1)(1+s)^2}}$$

$$= \frac{1}{s} \left[ \frac{s(5s+1)(1+s)^2}{s(5s+1)(1+s)^2 + K_1(2s+1)} \right] + \frac{6}{s^2} \left[ \frac{s(5s+1)(1+s)^2}{s(5s+1)(1+s)^2 + K_1(2s+1)} \right]$$

$$e_{ss} = \lim_{s \rightarrow 0} s E(s)$$

$$= \lim_{s \rightarrow 0} s \left[ \frac{1}{s} \left[ \frac{s(5s+1)(1+s)^2}{s(5s+1)(1+s)^2 + K_1(2s+1)} \right] + \frac{6}{s^2} \left[ \frac{s(5s+1)(1+s)^2}{s(5s+1)(1+s)^2 + K_1(2s+1)} \right] \right]$$

$$e_{ss} = 0 + \frac{6}{K_1} = \frac{6}{K_1}$$

$$e_{ss} < 0.1 \quad \therefore 0.1 = \frac{6}{K_1}, \quad K_1 = \frac{6}{0.1} = 60$$